



SIEMENS

Steam Turbine

Life time calculations and life limitings factors

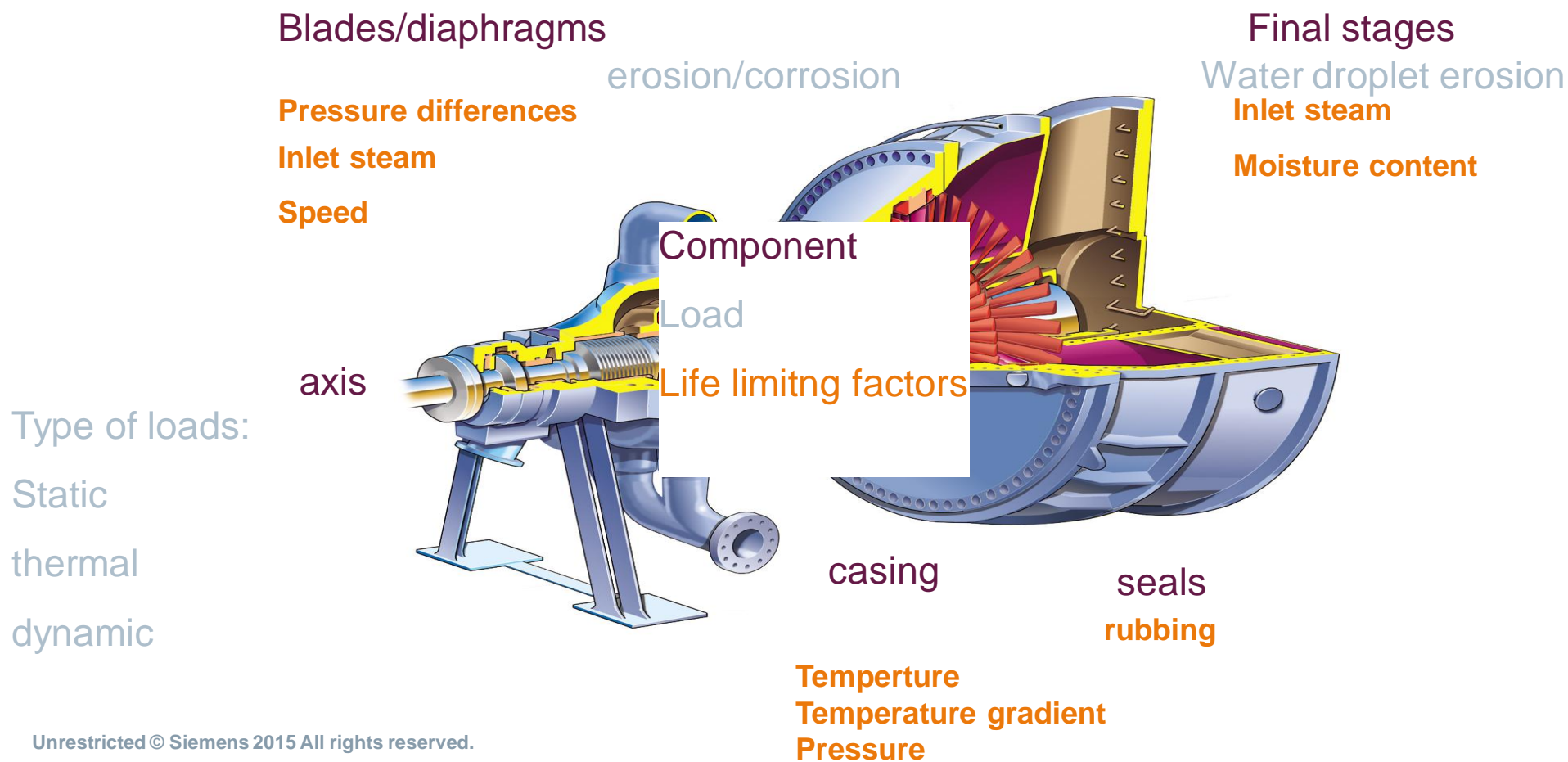
Göteborg 2017

AGENDA



- **Intro - Life time calculation (theory)**
- Steam turbine start up and shut down
 - Limitations
 - Improvements real examples
- Life limiting factors during turbine life time
 - Life time assessment (LTA)

Life time calculations Limitations/Influence



Life time calculations

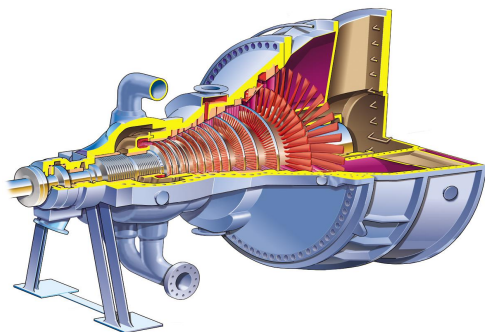
Basic criteria

The steam turbine should be able to handle all loads that it might be exposed to during the design lifetime without inaplicable deformations and brott apperas.

Life time calculations

Load profile

**Thermal transients,
Static load
and
Dynamic load**



Safety and life time aspects

Typical load profil

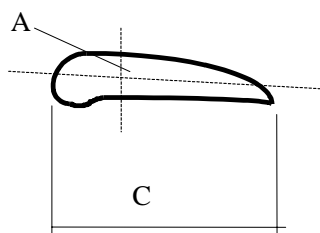
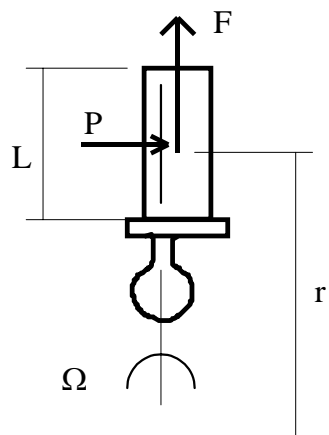
- Start – full load - stop
 - ~ 2000 - 10000 cycles
 - 100 000 – 200 000 EOH
 - Metal temperaturer -30 - 585 C
- Vibrationer

Real loads verified by

- Lab test
- Prototyp testing
- Operational experience (RDS)
- LTA

Life time calculations

Static load on a turbine blade



För en skovel enligt figur fås spänningen på grund av
Centrifugal last

$$\sigma_c = F / A \sim \rho L r \Omega^2$$

böjspänning på grund av statisk gaslast

$$\sigma_b = M_b / W_b \sim P L / (z C^3)$$

Termisk last

$$\sigma_t \sim \alpha \Delta T$$

Ω = rpm, P = gas load and z = no. of blades.

T = temperature, α = thermal elongation

Life time calculations

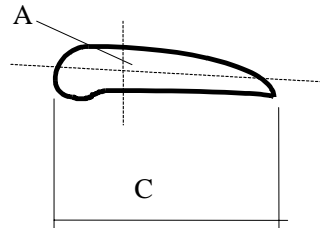
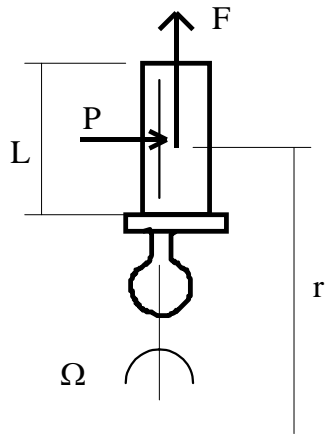
Dynamic load on a turbine blade

The load on the blade can be general described as

$$P(t) = P_m + P_a \sin(\omega t)$$

$$\sigma_a = S H_n A \pi / \delta \sigma_m = D_n \sigma_m$$

Dynamic factor depends on type of stage design



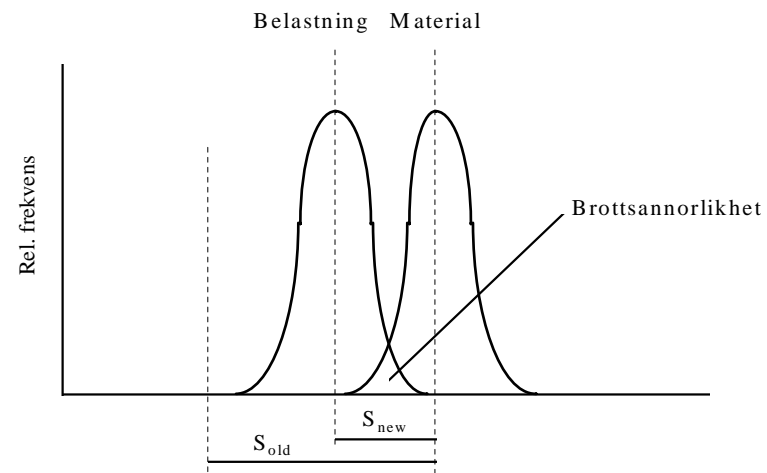
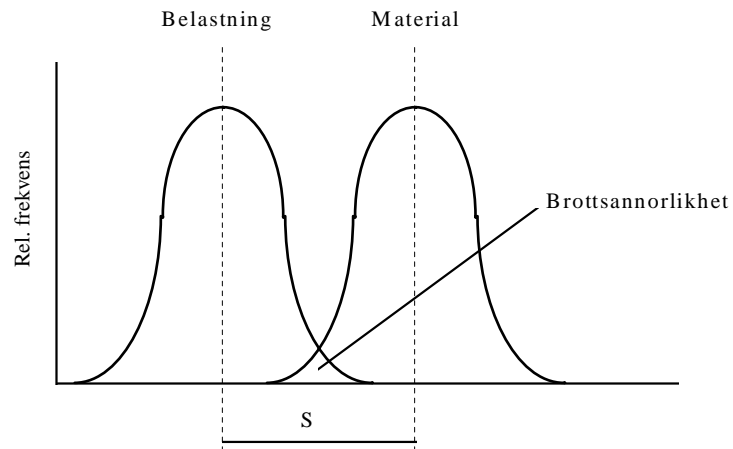
Life time calculations

Statistisk brottsannorlikhet

Uncertainties

- Load
- Calculation tool
- Manufacturing

+ Material data



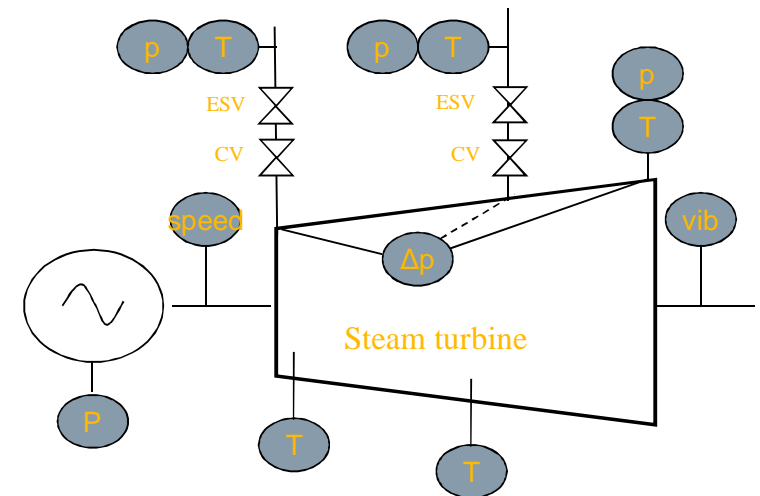
How can turbine operation be controlled?

What needs to be controlled?

Power/frequency control
with admission steam valves

Steam parameter control

Operation limitations for turbine protection



P: Power; p: pressure, T: temperature

AGENDA



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Steam turbine start up and shut down

Typical number of starts per year

We thought

Base load

- 3 cold starts per year
- 6 warm starts per year
- 22 hot starts per year

Peak load

- 5 cold starts per year
- 50 warm starts per year
- 200 hot starts per year

Solar plants

- 20 cold starts per year
- 340 warm starts per year
- 20 hot starts per year

Reality

Base load

- 3 cold starts per year
- 6 warm starts per year
- 22 hot starts per year

Peak load

- 50 cold starts per year
- 125 warm starts per year
- 100 hot starts per year

Solar plants

- 120 cold starts per year
- 200 warm starts per year
- 5 hot starts per year

Steam turbine start up and shut down

Start up category

Start category (cold, warm, hot) is chosen based on casing temperature

The start up category depends on stand still time between starts, surrounding temperature, status on insulation etc

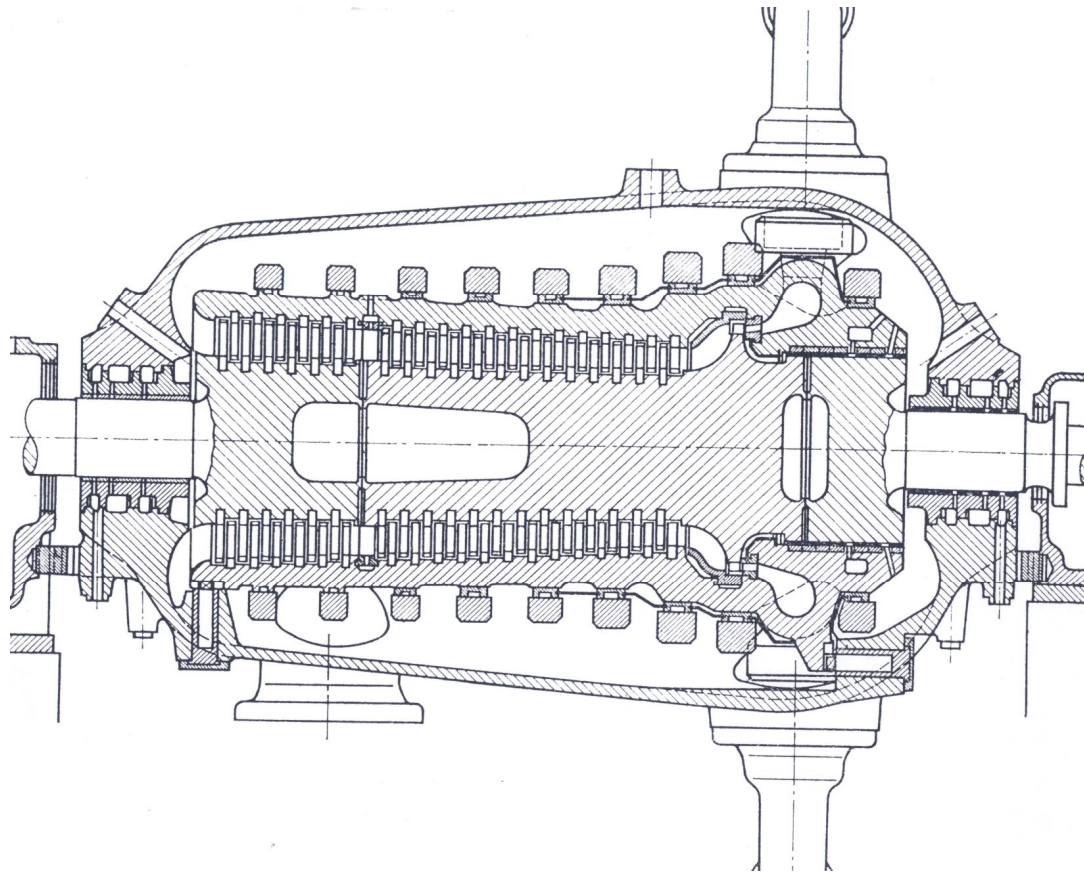
Steam turbine start up and shut down

Start up- What are the challenges

Large centrifugal loads (static)	LP
Bending/torsion loads	LP
Large thermal loads (LCF, creep)	HP, IP
Material properties	HP, IP
Unsteady steam loads (HCF)	HP, IP, LP
Stress concentration in notches	HP, IP, LP
Corrosion	LP
Erosion	LP
Oxidation	
Rubbing	IP, LP

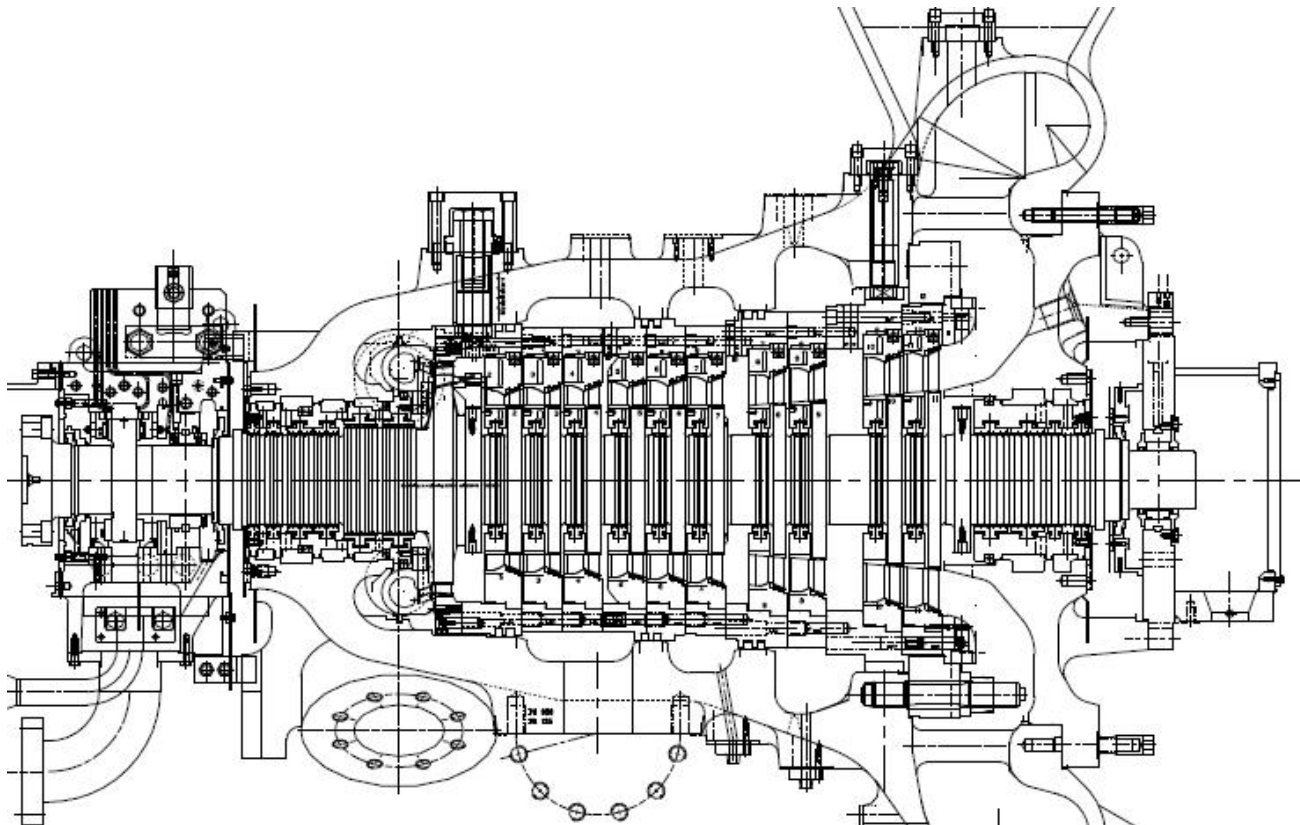
Steam turbine start up and shut down

High pressure reaction steam turbine with control stage



Steam turbine start up and shut down

High pressure impulse steam turbine with control stage

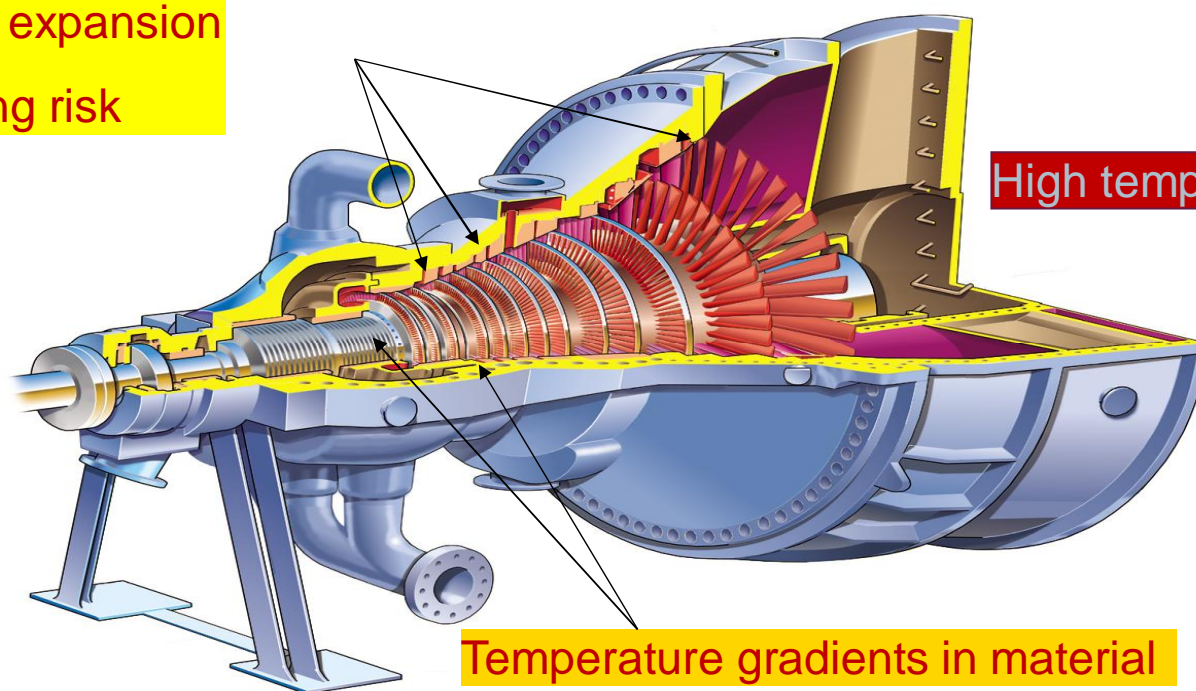


Steam turbine start up and shut down

Specific limitations

Start up = operation with specific limitations

Thermal expansion
-> rubbing risk



High temperature in exhaust

Temperature gradients in material

Thermal stress

Material fatigue

Steam turbine start up and shut down

Steam turbine start up time

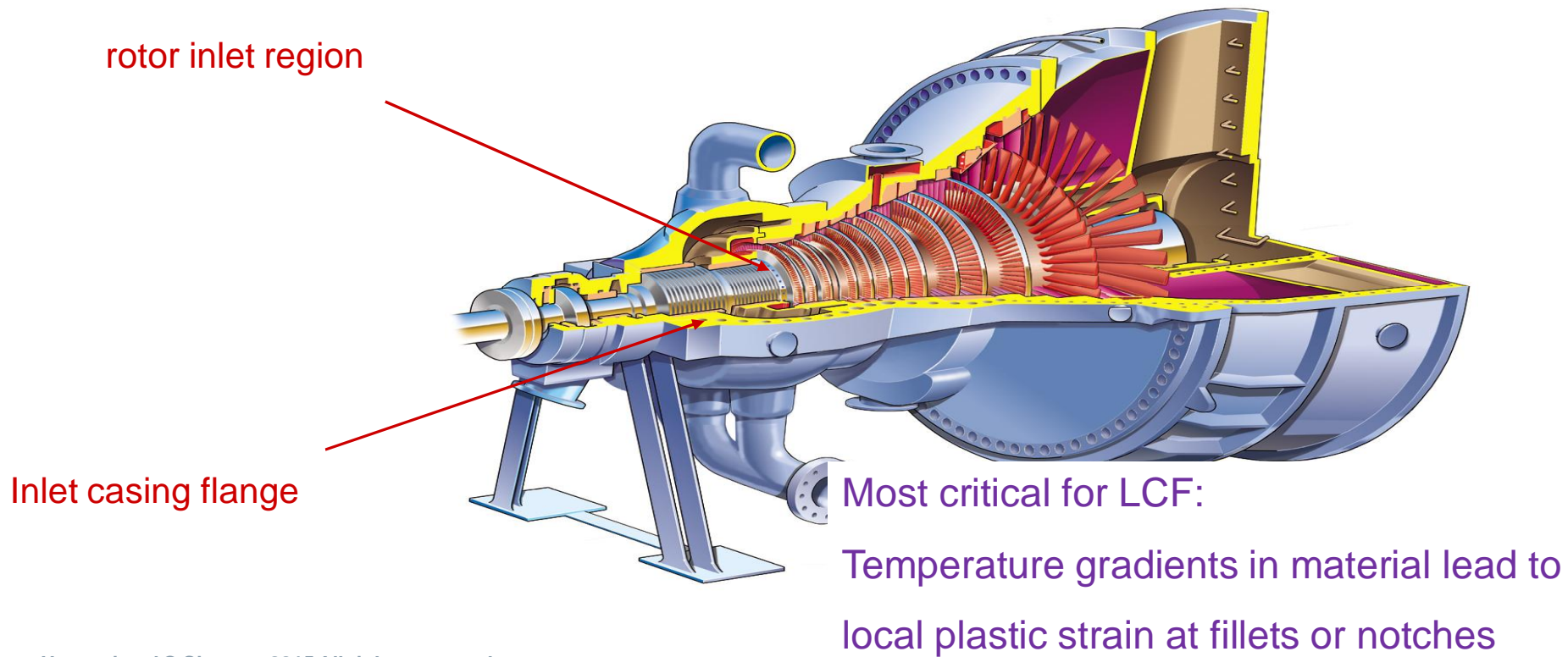
Limited by allowed thermal stresses and clearances

Dependent on

- Inlet steam data
- Control concept
- Turbine size and design
- Material

Steam turbine start up and shut down

Which parts are most critical for thermal fatigue during start ?



Steam turbine start up and shut down

Concept of linear life consumption

$$\sum \frac{N}{N_F} + D_{Creep} \leq 1 - (\text{safety margin})$$

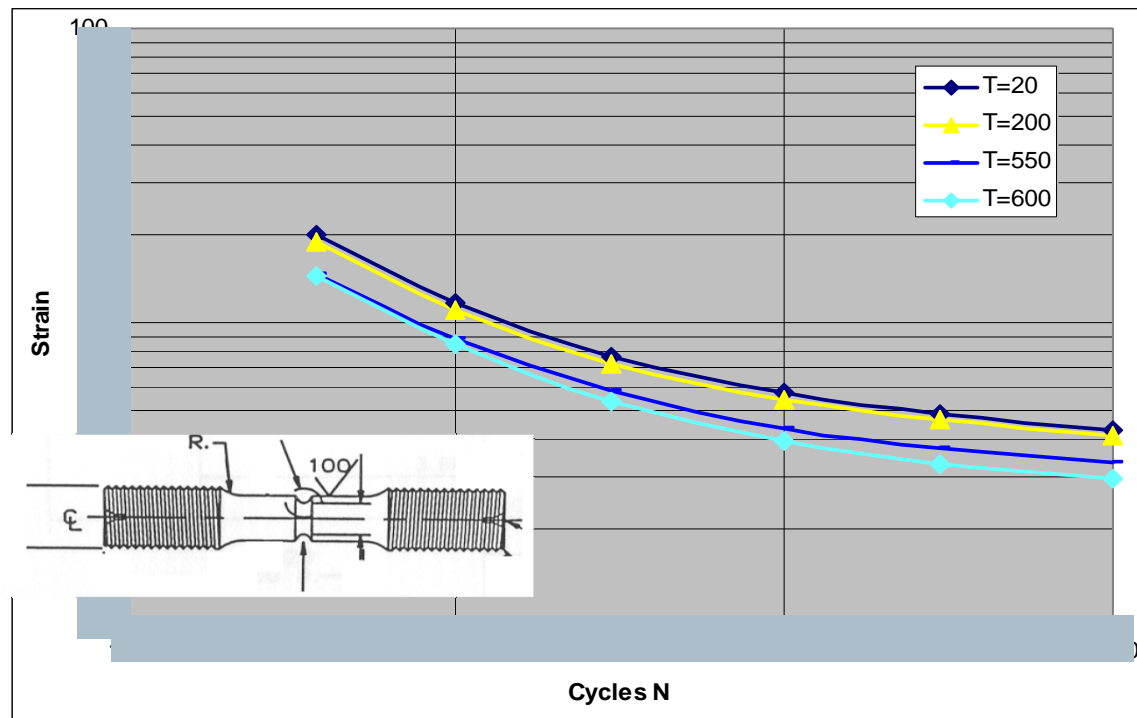
N: Number of cycles

N_F: Number of cycles until crack initiation due to LCF (temperature dependent)

N/N_F: LCF –damage (Palmgren –Miners linear damage rule)

D_{creep}: Creep damage (for temperatures T > T*)

Steam turbine start up and shut down LCF material data (example)



This is not the material situation in reality!

Estimated under well
defined conditions

- Load direction
- Load cycles
- Geometry
- Temperature ...

Statistical evaluation

Steam turbine start up and shut down

Simplified thermal stress and strain during start

General

$$\varepsilon = \beta \cdot \Delta T$$

β : thermal expansion coefficient

Radial strain

- expansion outwards
- largest stress in center of body (tensile)
- zero stress at outer wall

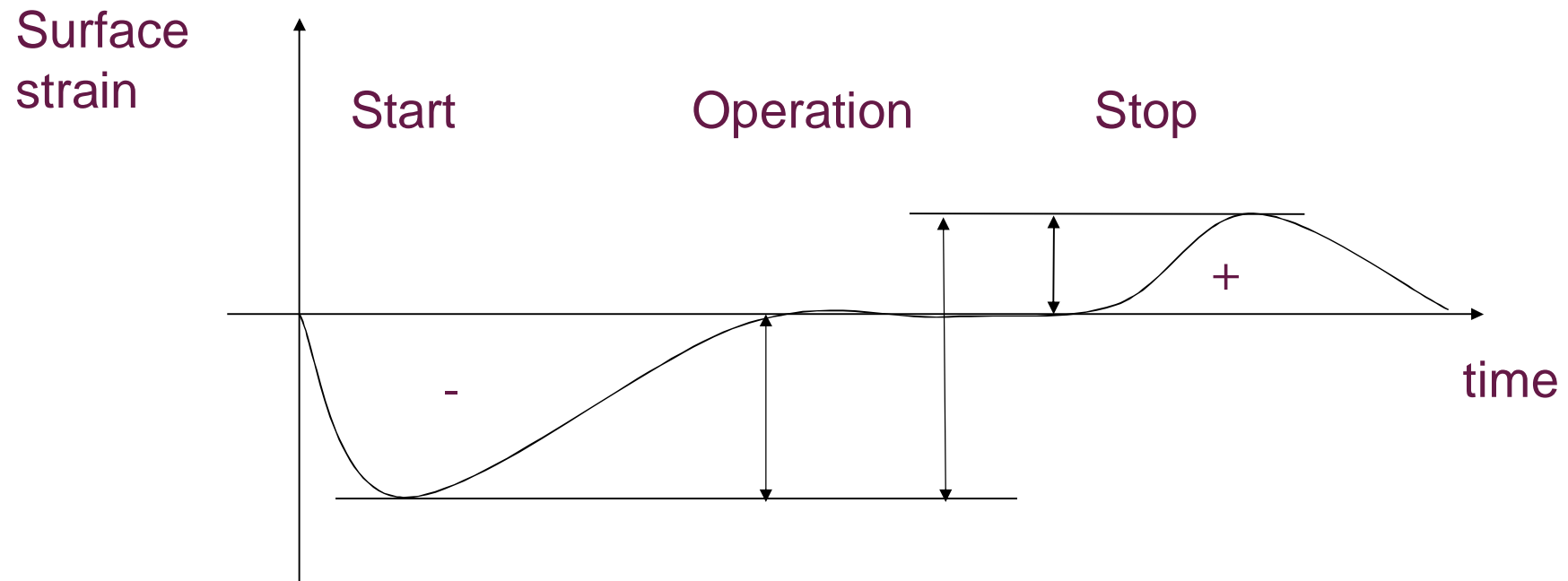
Tangential strain

- strain prohibited in outer region → compressive stress
- Less strain in the colder inner region → tensile stress

Axial strain

- Strain prohibited in outer region → compressive stress
- Less strain in the colder inner region → tensile stress

Steam turbine start up and shut down Start / stop cycle



Steam turbine start up and shut down

Simplified: Axial stress model

$$\sigma = E \cdot \varepsilon = E \cdot \frac{\beta}{1 - \nu} \cdot \Delta T$$



An allowed stress can be expressed as an allowed temperature difference:

$$\Delta T = T_i - T_m < \Delta T_{allowed}$$

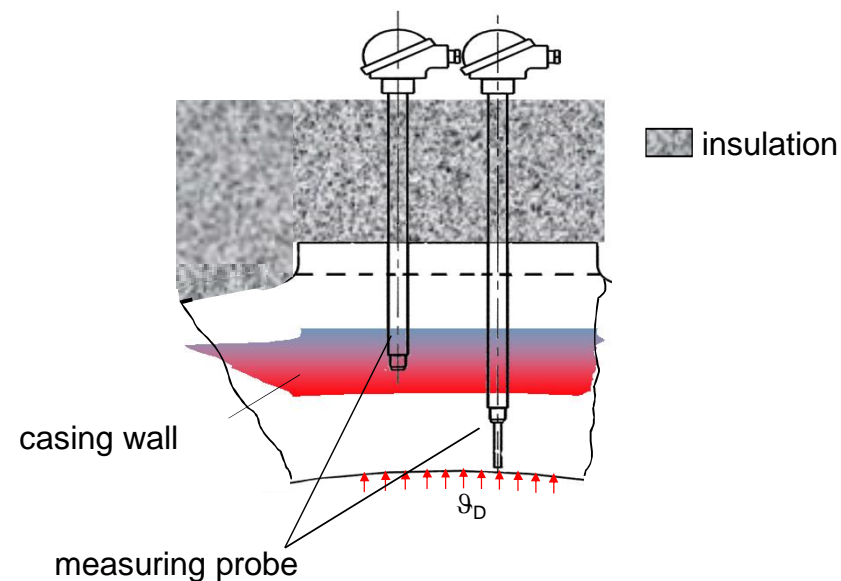
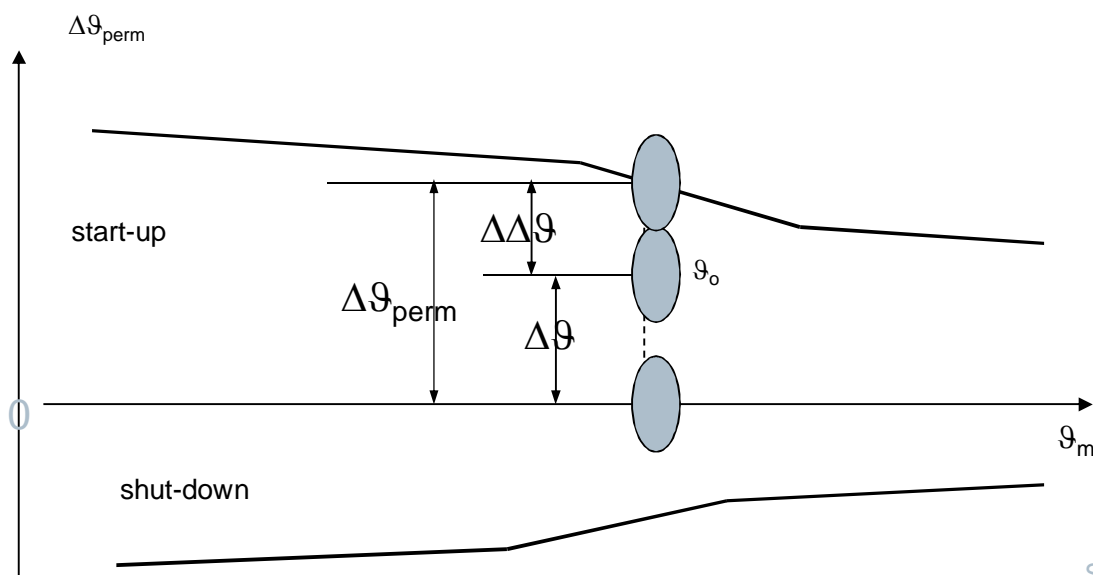


Temperature distribution in
Material needed!

Steam turbine start up and shut down

Thermal stress control

- measure temperature differences for thick walled components
- simulation for shaft temperatures
- influence on automatic start-up program



Source: internal communication with LSU, 2013

Examples: Design to reduce thermal stresses

Reduce material thickness (Barrel design)

Preheat , keep turbine warm

Optimize start up transients (stress evaluator)

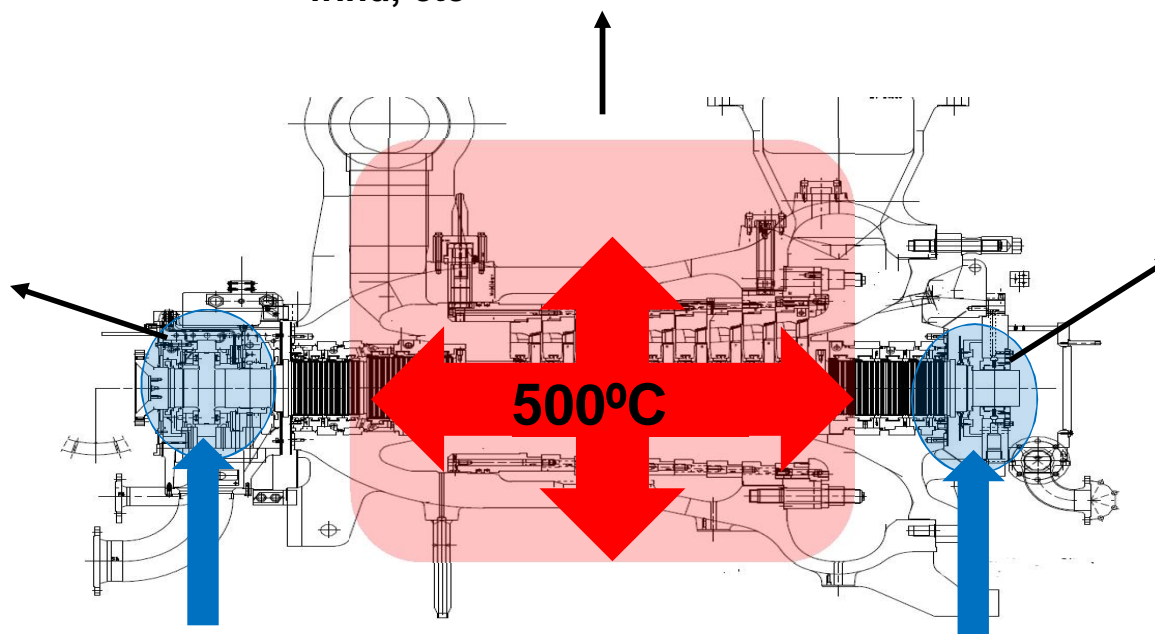
Optimize material use (locally better material)

Improving start up time Heat losses HP

Where does the heat go?

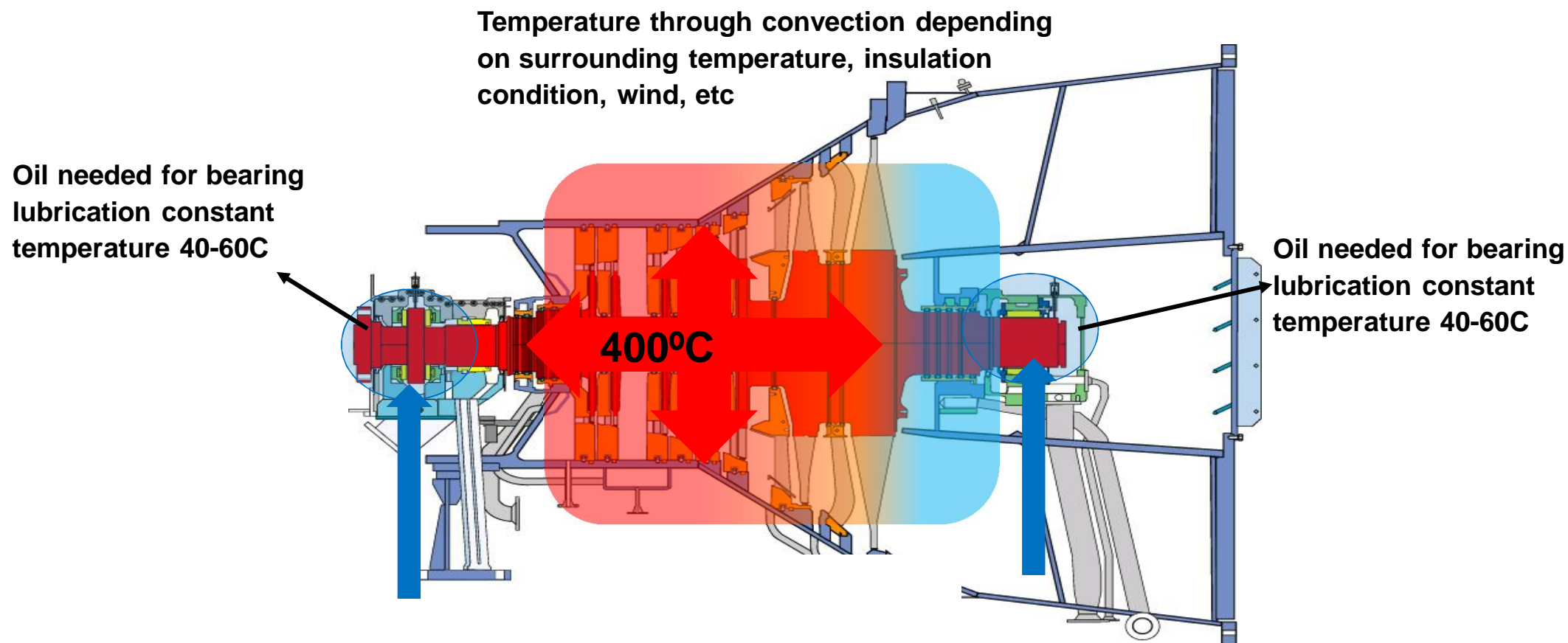
Temperature through convection
depending on surrounding
temperature, insulation condition,
wind, etc

Oil needed for bearing
lubrication constant
temperature 40-60°C



Oil needed for bearing
lubrication constant
temperature 40-60°C

Improving start up time Heat losses LP



Improving start up time What is needed?

Heating blankets casing

Gland steam

Axial displacement transmitters

Improving start up time

Heating blankets

- 5 Zones that will keep the turbine casing warm
- Gland steam to keep the rotor warm
- Supervision of thermal elongation between rotor and casing:
 - No rubbing between stator and rotor
 - Equal temperature between casing and rotor

Improving start up time

Heating blankets

- 4 Zones that will keep the turbine casing warm
- Gland steam to keep the rotor warm
- Supervision of thermal elongation between rotor and casing:
 - No rubbing between stator and rotor
 - Equal temperature between casing and rotor

Improving start up time Heating blankets

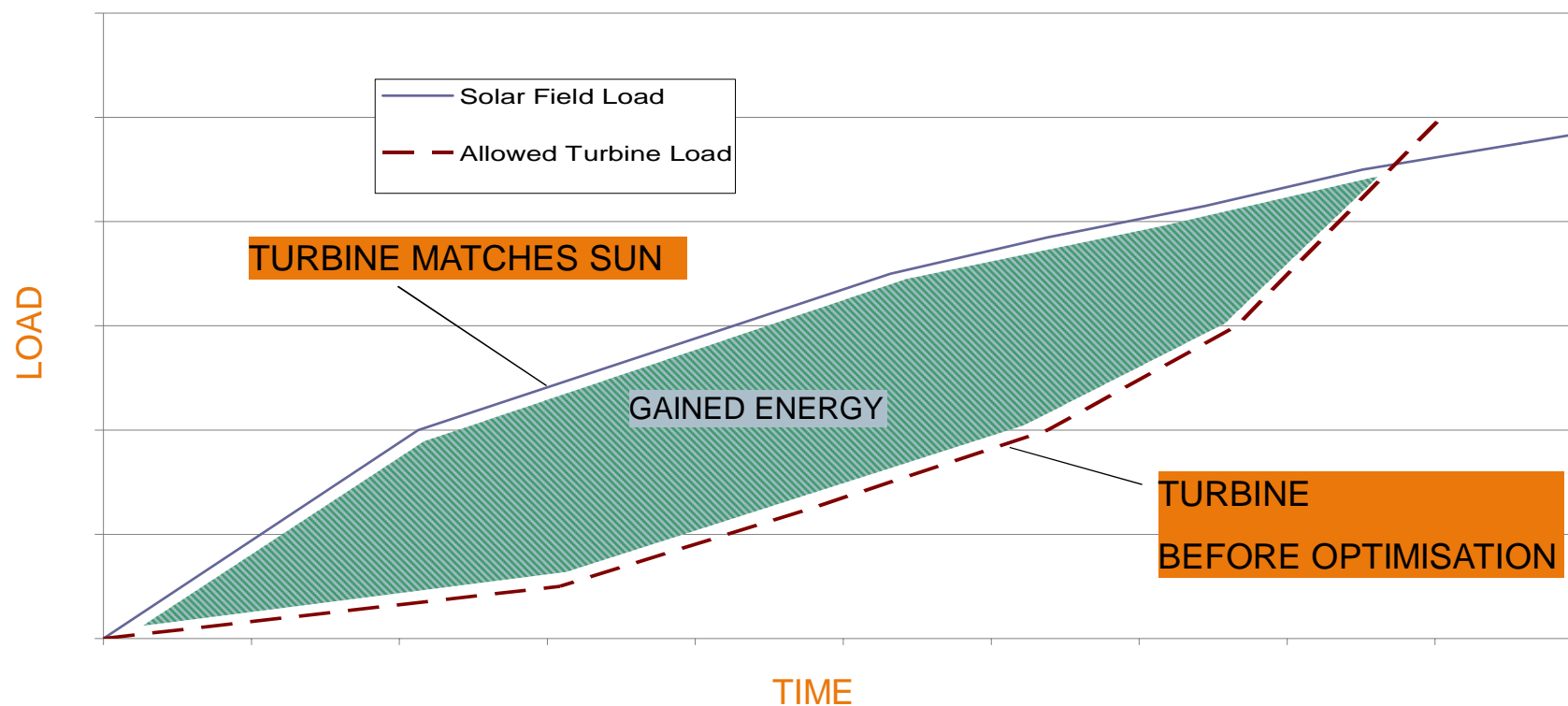


Improving start up time Example

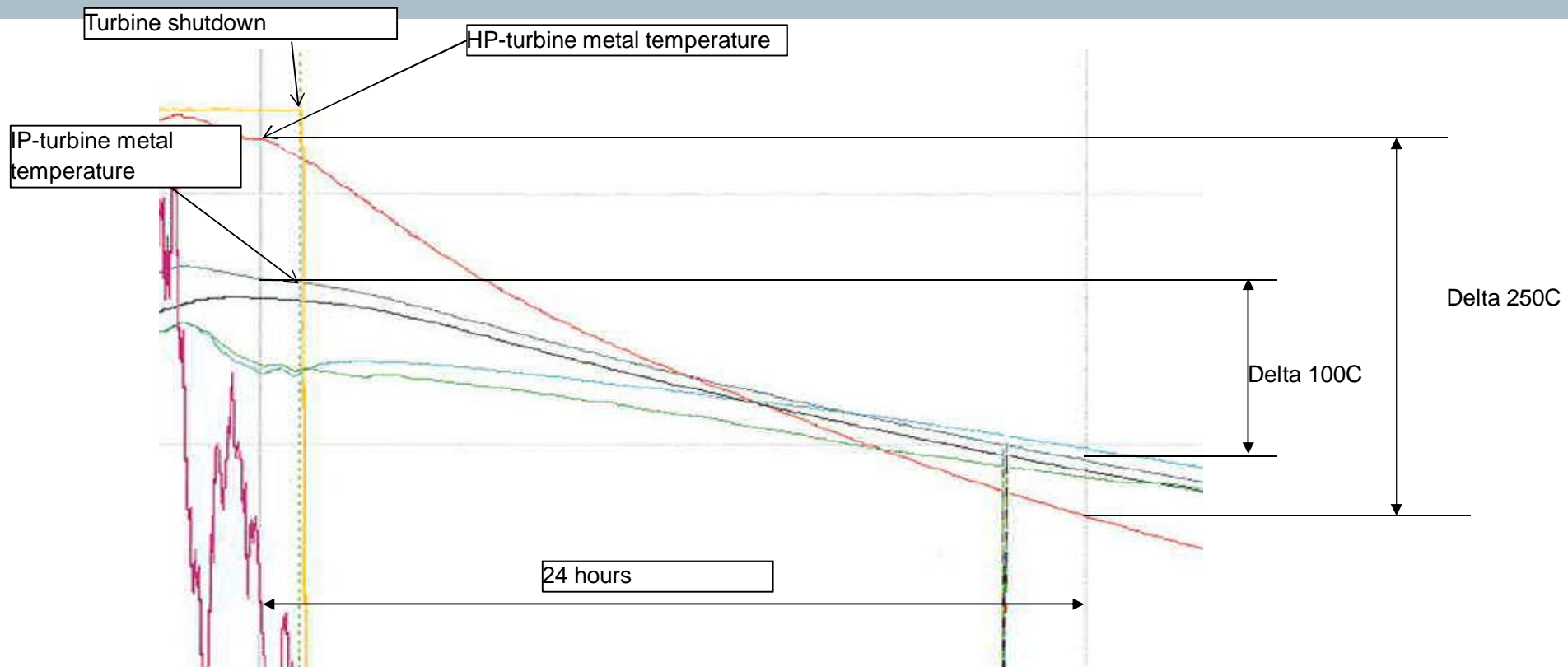
CSP (Concentrated Solar Power)



Improving start up time Why?

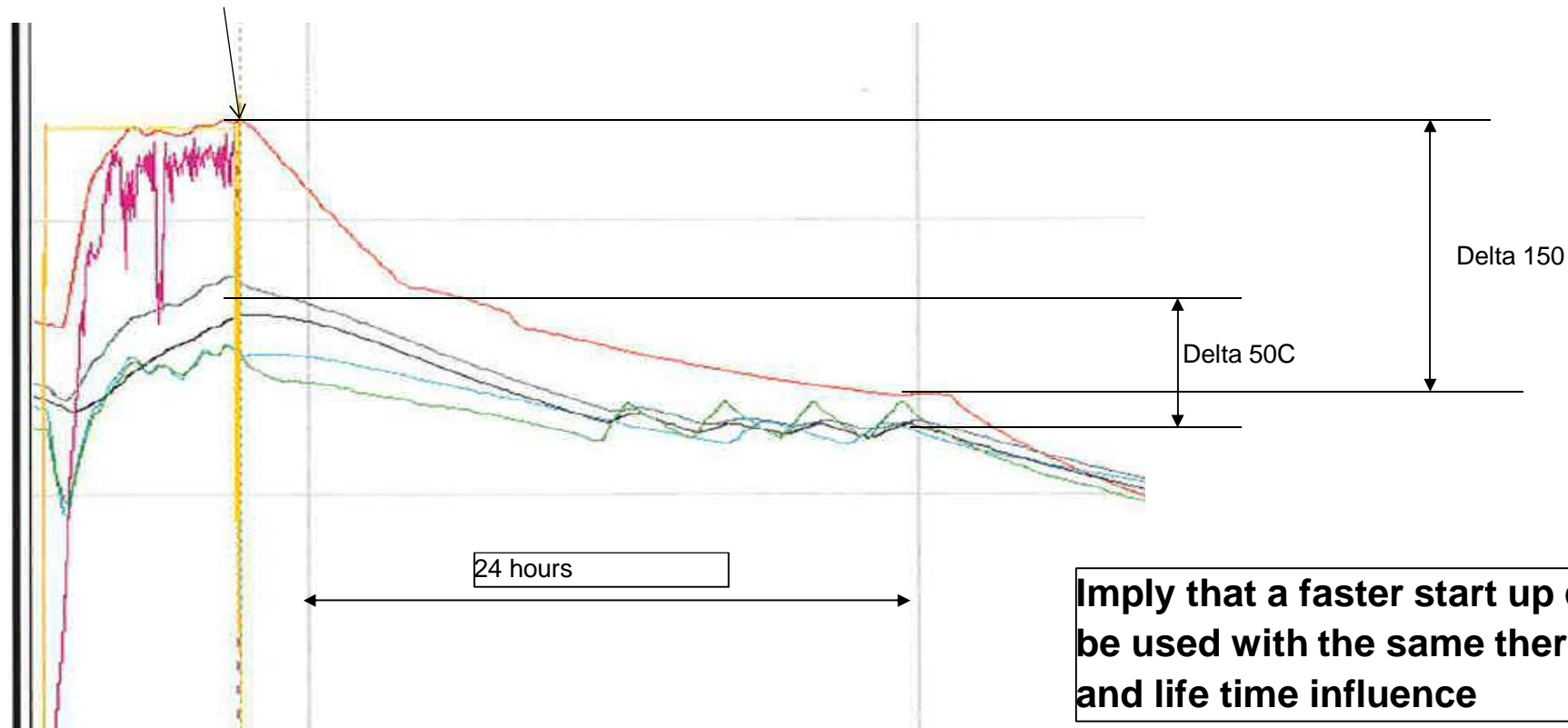


Improving start up time Before installation



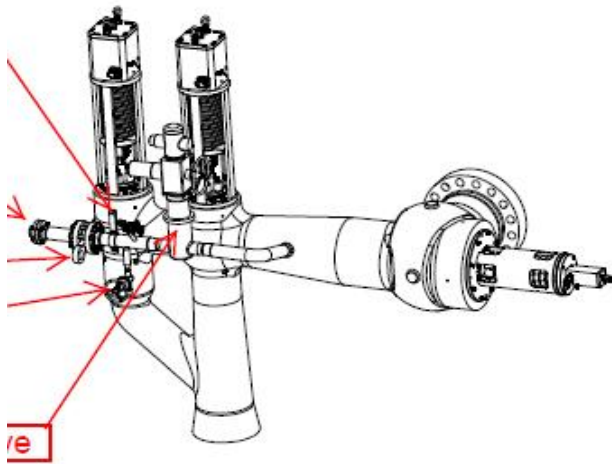
Improving start up time After installation

Turbine shutdown



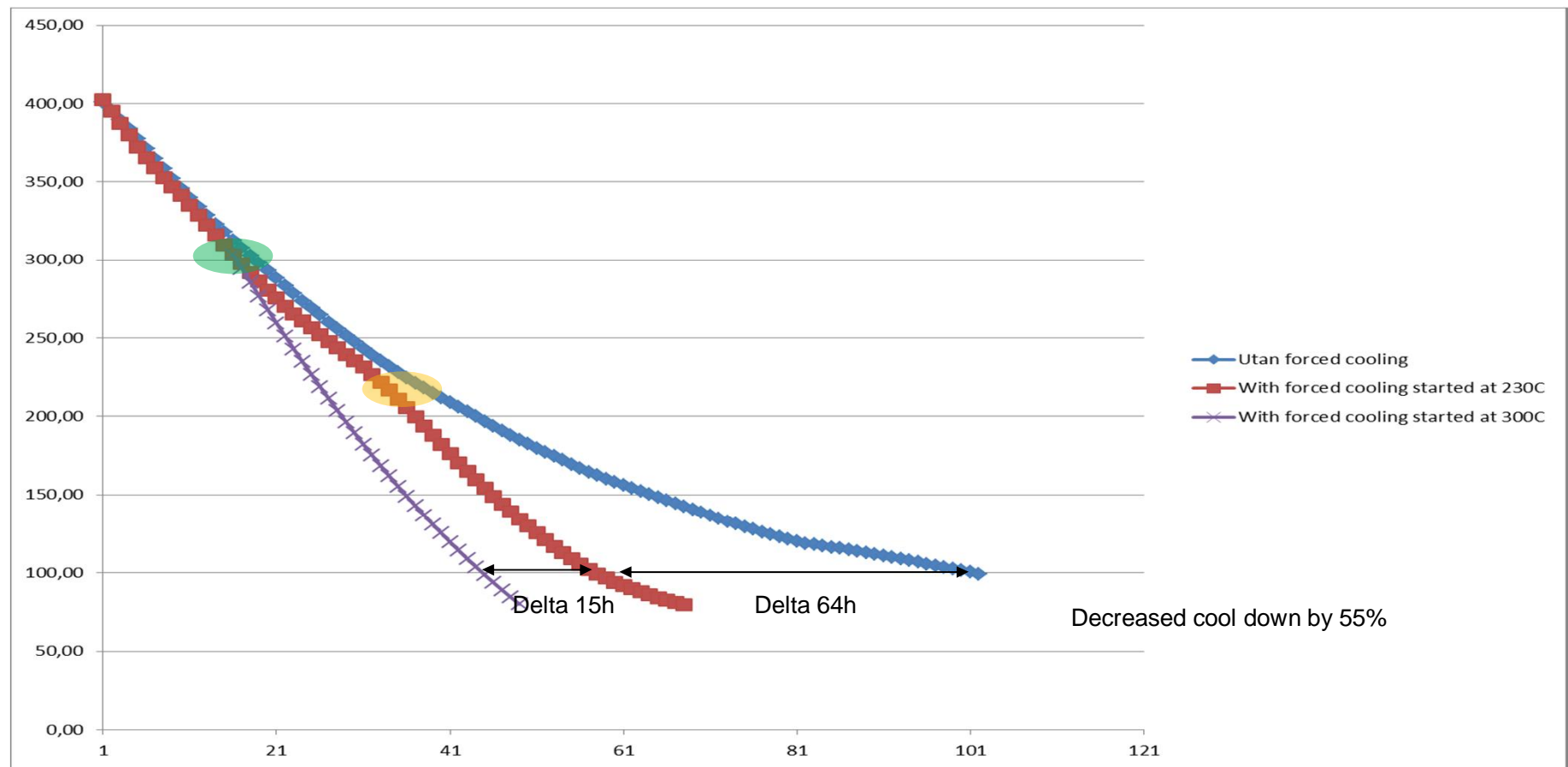
**Imply that a faster start up curve can
be used with the same thermal stress
and life time influence**

Forced cooling What is needed



Axial displacement transmitters

Forced cooling Out come





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Thank you for your attention!

End