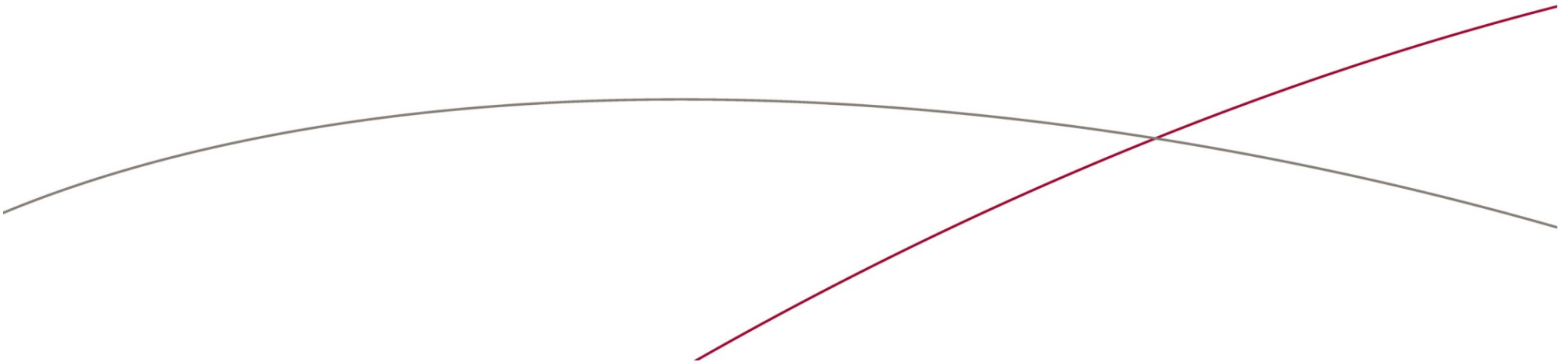




KME 707

Improved steam turbine design for optimum efficiency and reduced cost of ownership

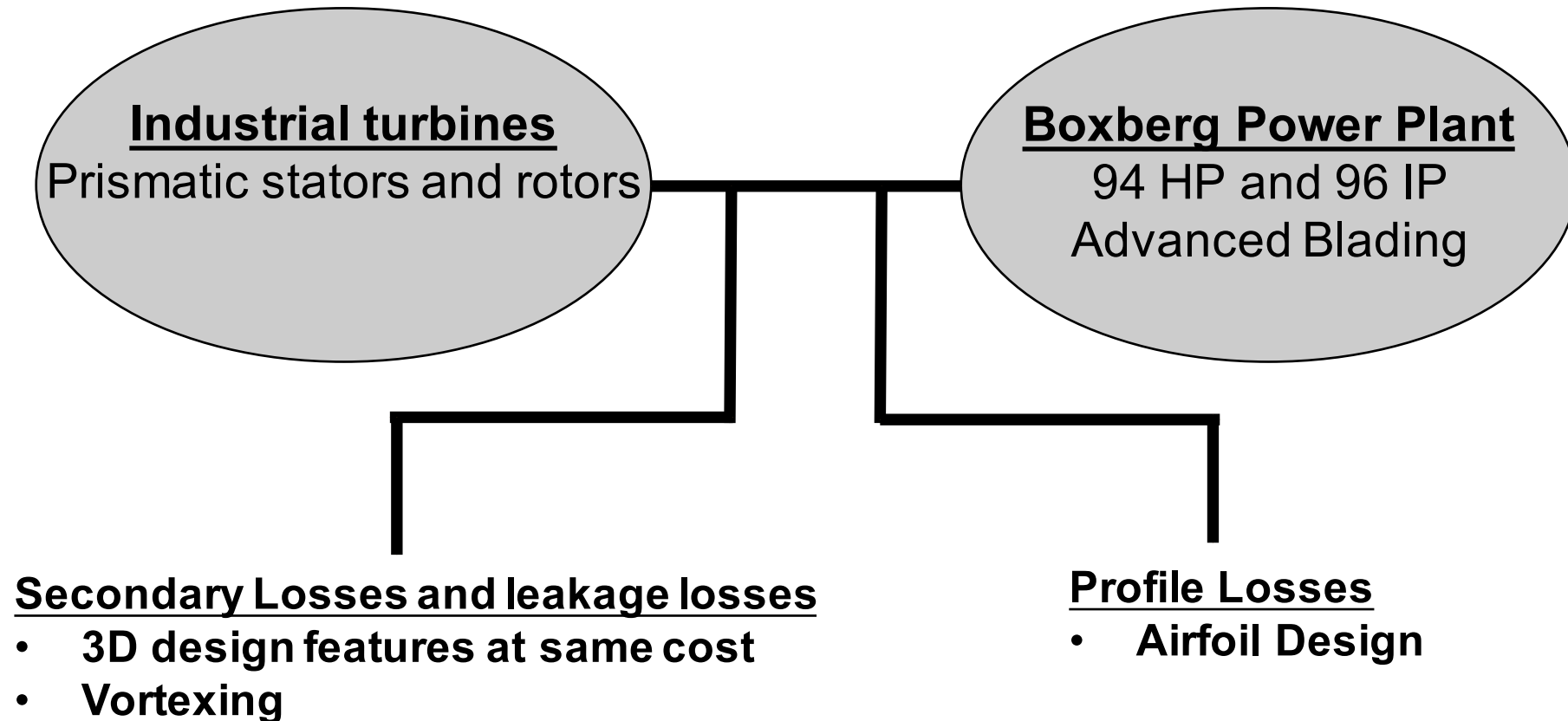


Agenda

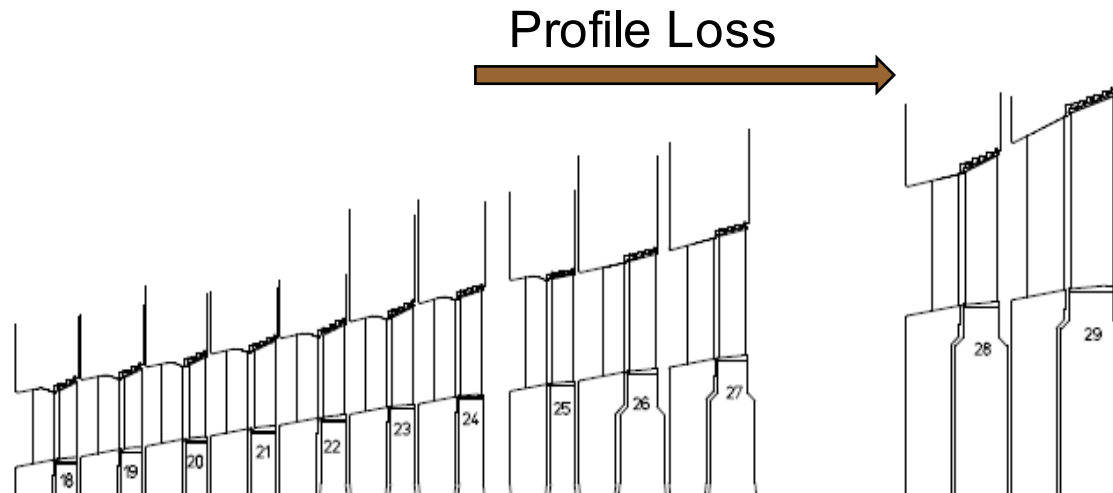
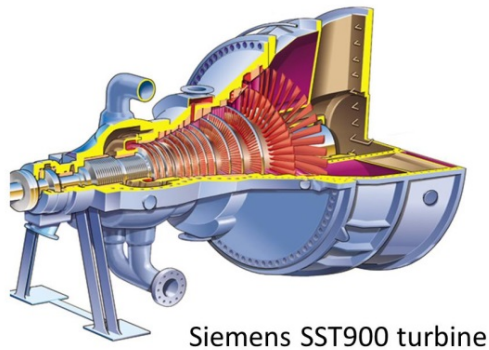
- Background
- Case Study
- Work Done
- Publication Summary
- Short Stages
- CFD model setup
- Future work – rotor modification



Background / Motivation



Case study

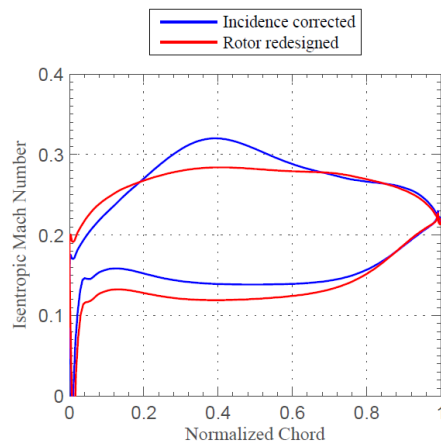
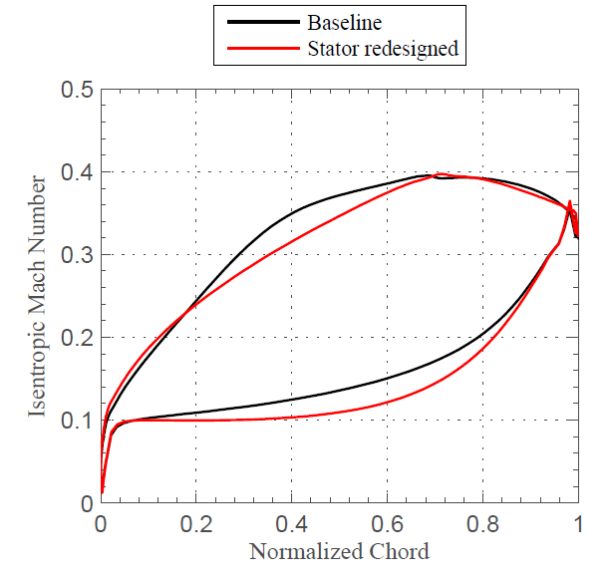
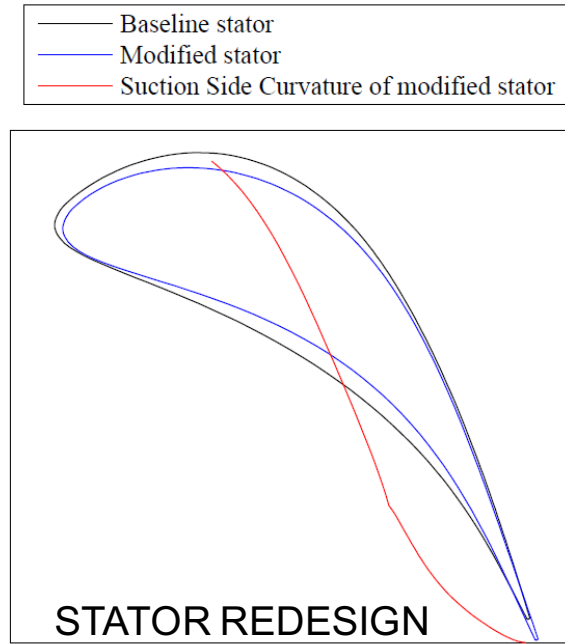
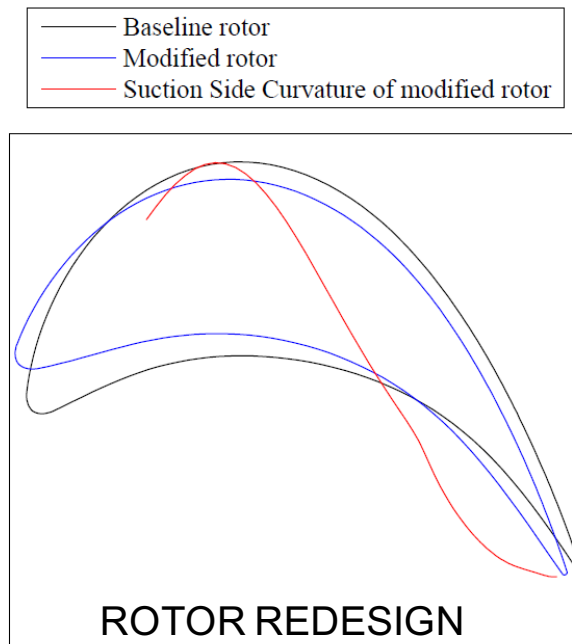


Concepts Attempted :

- Vortexing – Stator and Rotor
- Airfoil redesign
- Flow path modification
- Lean



Airfoil Design – High aspect ratio stages



Modifications	Y_{stator}	Y_{rotor}	$\Delta\eta$
Rotor incidence correction	0.0	0.6	0.23
Rotor redesign	0.0	0.8	0.26
Stator redesign	0.14	0.0	0.37



Summary of publications

ASME GTINDIA2014

GTINDIA2014-8175

Reduction In Secondary Losses In a Turbine Cascade Using Contoured Boundary Layer Fence

TURBO EXPO 2015

GT2015-43235

Vortexing Methods To Reduce Secondary Losses In A Low-Reaction Industrial Turbine Vane

ASME GTINDIA2015

GTINDIA2015-1221

Influence of Compound Lean on an Industrial Steam Turbine stage

TURBO EXPO 2016

GT2016-57138

Efficiency Improvements in an Industrial steam turbine stage – Part I

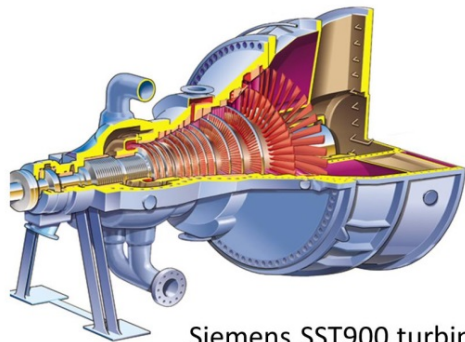
TURBO EXPO 2016

GT2016-57140

Efficiency Improvements in an Industrial steam turbine stage – Part II

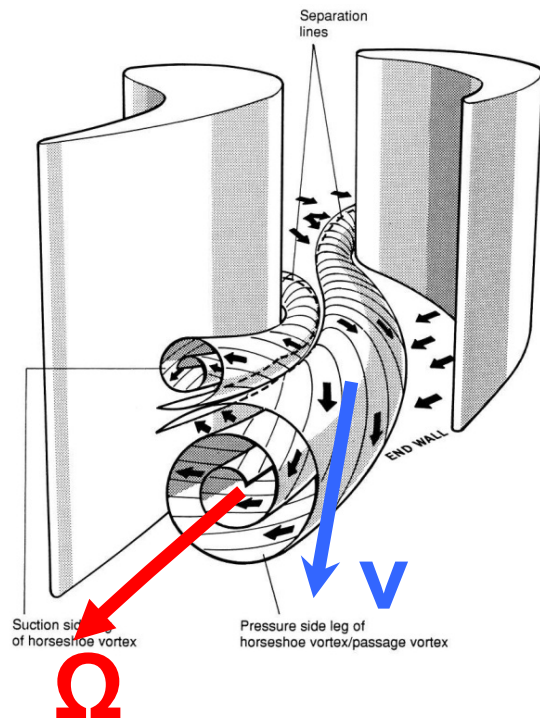
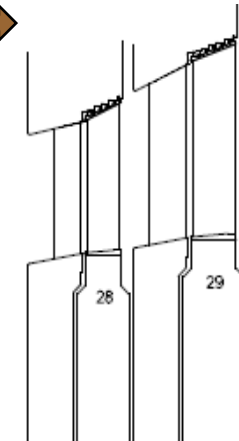
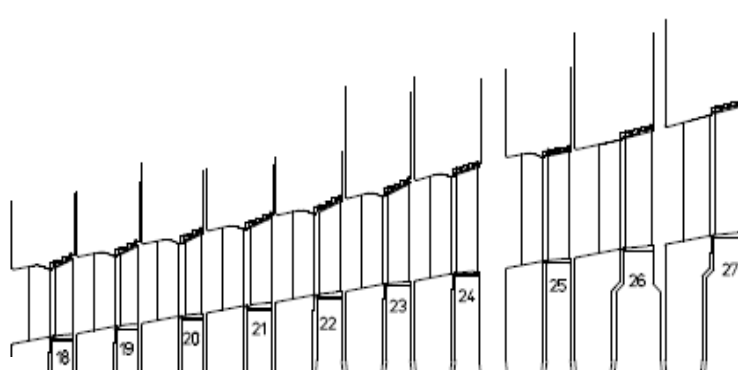


Case study

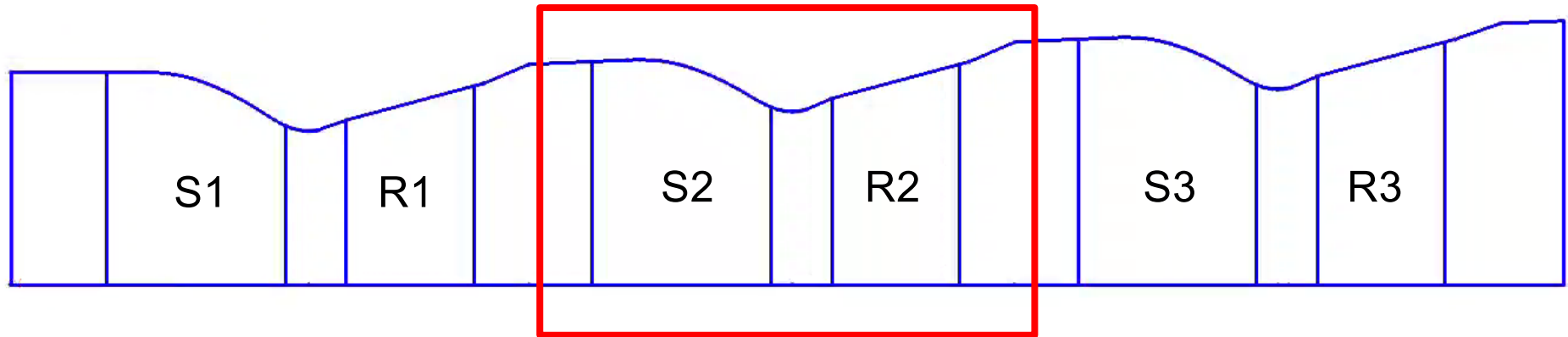


Secondary Loss

Profile Loss



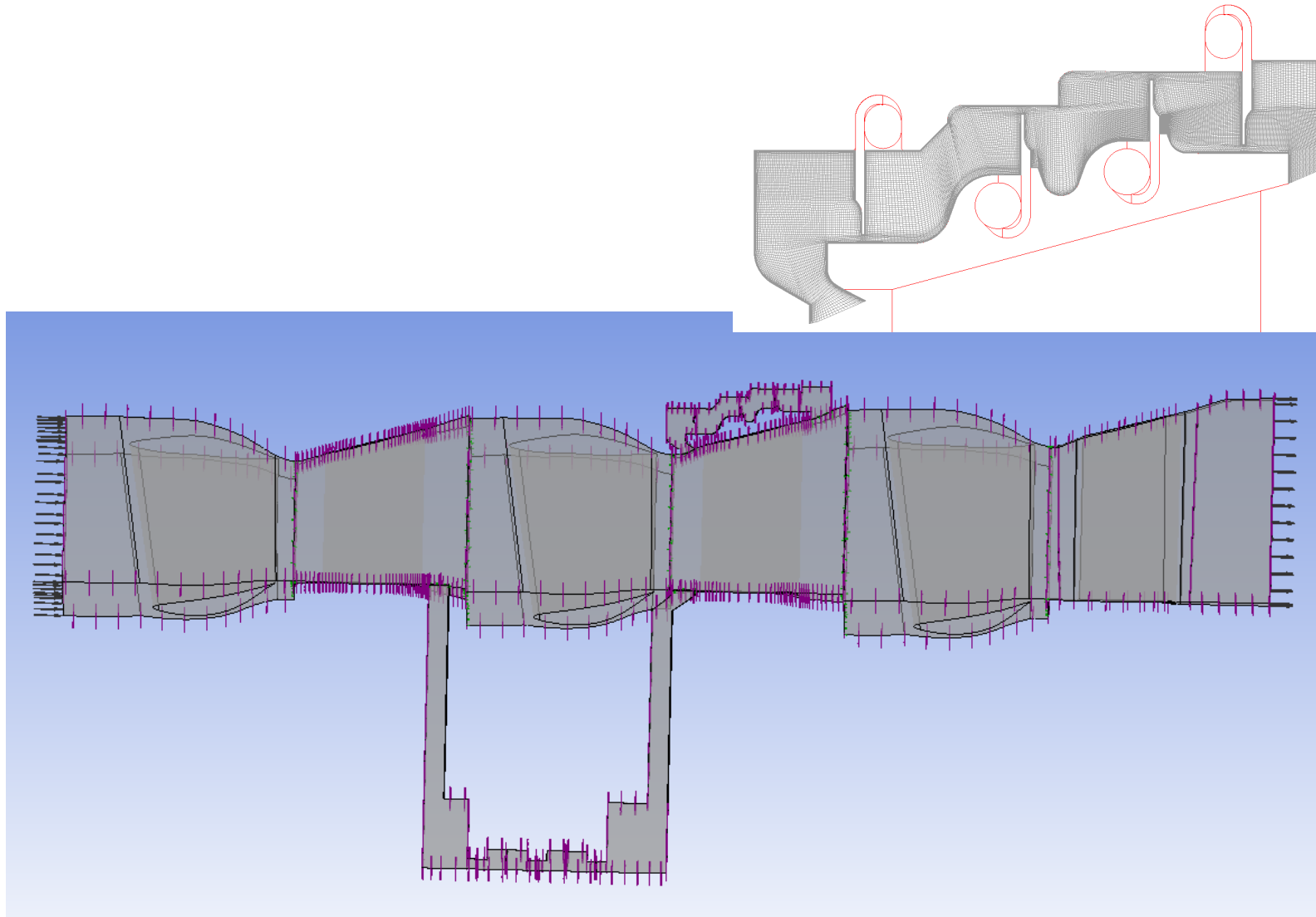
SST900 - HP



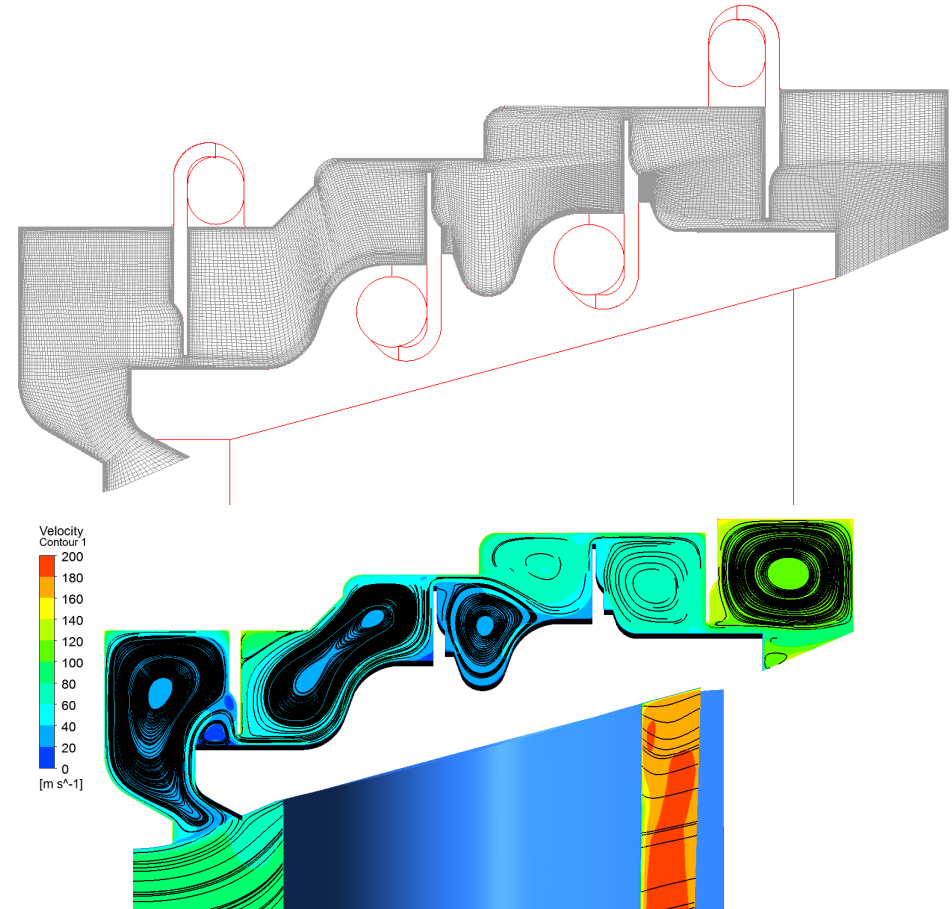
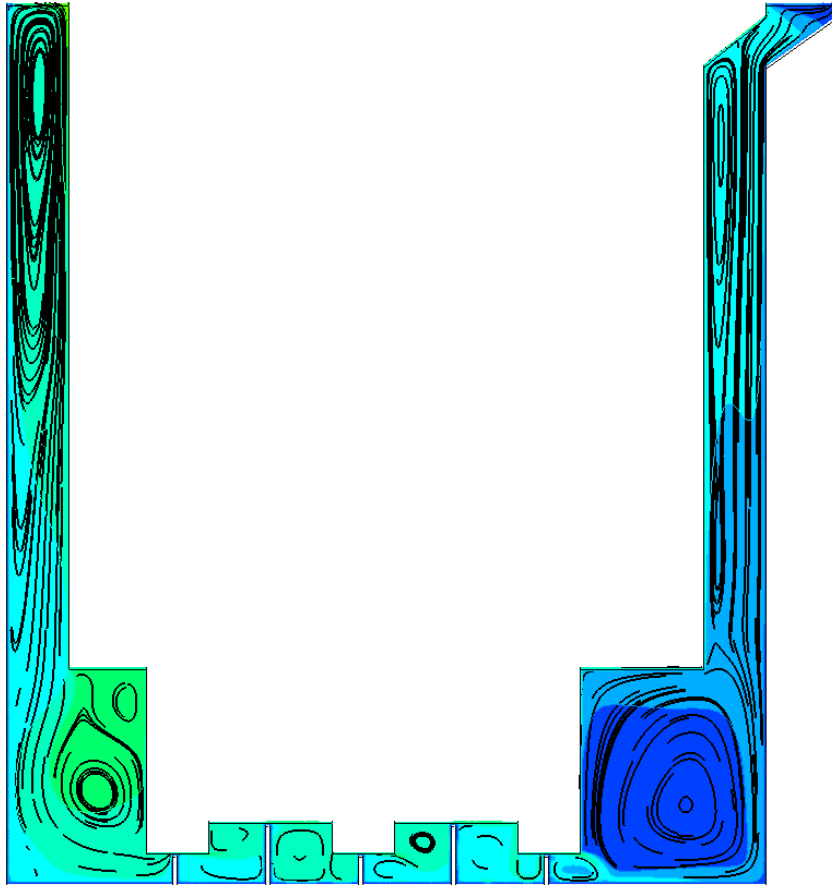
No of blades	32	64	32	64	32	65
pitch / chord	0.761	0.802	0.768	0.810	0.777	0.807
Relative Exit Mach	0.367	0.245	0.378	0.249	0.385	0.257
Reynolds number	9.64×10^6	3.06×10^6	9.2×10^6	2.88×10^6	8.65×10^6	2.75×10^6
Aspect ratio (axial)	1.2	1.3	1.25	1.46	1.37	1.64
Work Coefficient	1.64		1.64		1.64	
Flow Coefficient	0.33		0.32		0.316	
Reaction	0.224		0.22		0.23	



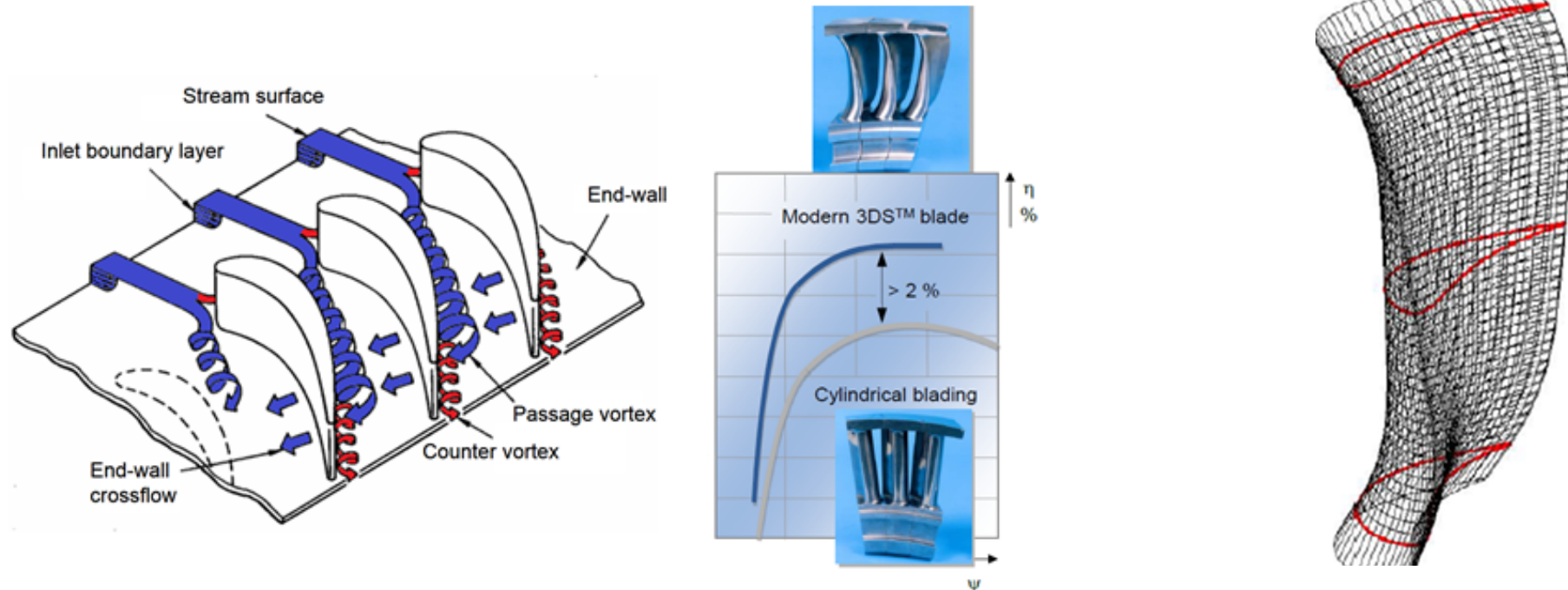
CFD Model setup



Leakage Flows modelled



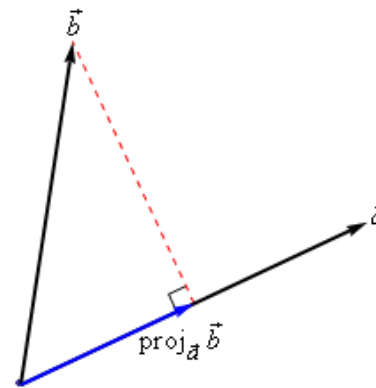
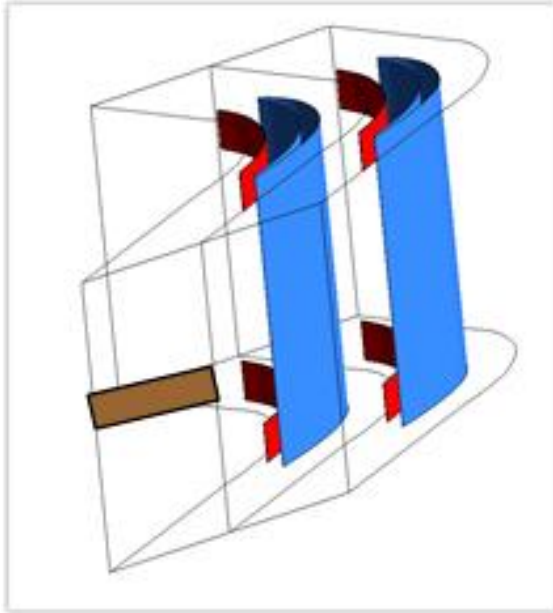
Future work - Rotor



- ❖ Rotor Modifications for maximum benefit
- ❖ Stator Modifications corresponding to Rotor



Secondary loss - quantification



$$\text{proj}_{\vec{a}} \vec{b} = \frac{\vec{a} \cdot \vec{b}}{\|\vec{a}\|^2} \vec{a}$$

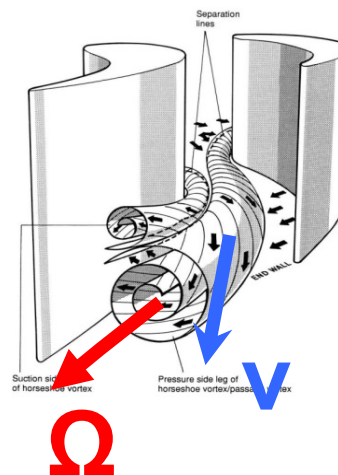
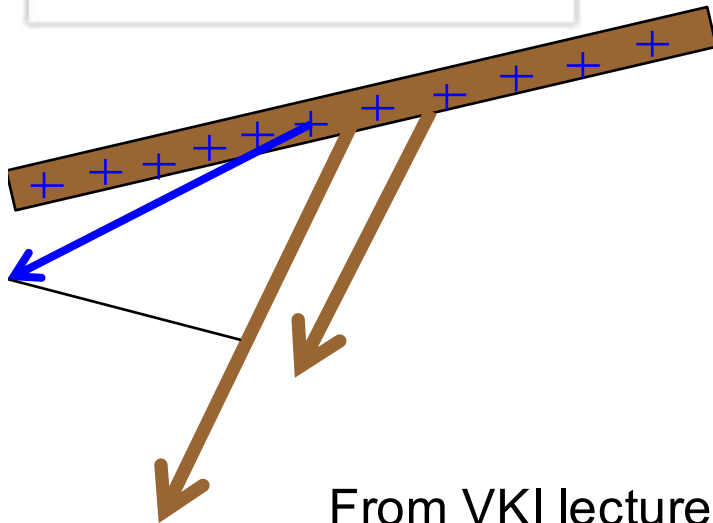
Corral et al.

$$CSKE = \frac{(v_i - v_{pi})^2}{v_{\text{exit}}^2}$$

$$v_{pi} = \frac{v_i \cdot v_m}{v_m^2} v_m$$

$$H_i = \frac{|v_i \cdot \Omega_i|}{v_{\text{exit}}^2 / l_c}$$

$$SKEH = CSKE_i \cdot H_i$$



From VKI lecture series : Haller et al



Timelines

	2014											
Task	J	F	M	A	M	J	J	A	S	O	N	D
Month									1	2	3	4
Project start									M1			
Kick-off									M1			
Status report												
Work package 1												
Work package 2												
Academic's												
Final reporting1												
Project end												

	2015											
Task	J	F	M	A	M	J	J	A	S	O	N	D
Month	5	6	7	8	9	10	11	12	13	14	15	16
Project start												
Kick-off												
Status report		#1						#2				
Work package 1	D1	D2						D3				
Work package 2								M3				
Academic's								D4				
Final reporting1												
Project end												

Milstolpe	Månad
M1 Projektstart/kick-off	1
M2 Optimal bladprofil	4
M3 Licentiatavhandling – Srikanth Deshpande	12
M4 Vridningsfilosofier och optimal tredimensionell stackning	24
M5 Designfilosofi för optimal industriångturbin	30
M6 Doktorsavhandling – Srikanth Deshpande	36

Leverabel	Månad
D1 Rapport "Optimala 2D bladprofiler" – 2D koordinater	6
D2 Statusrapport #1	6
D3 Statusrapport #2	12
D4 Licentiatavhandling – Srikanth Deshpande	12
D5 Statusrapport #3	18
D6 Rapport "Vridningsfilosofier och tredimensionell stackning"	24
D7 Statusrapport #4	24
D8 Rapport "Designfilosofi industriångturbin"	30
D9 Statusrapport #5	30
D10 Doktorsavhandling – Srikanth Deshpande	36
D11 Slutrapport	36



THANK YOU

